

Thermal Model Calibration of a Squirrel-Cage Induction Machine

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Abstract—Accurate and efficient thermal simulations of induction machines are indispensable for detecting thermal hot spots and hence avoiding potential material failure in an early design stage. A goal is the better utilization of the machines with reduced safety margins due to a better knowledge of the critical conditions. In this work, the parameters of a two-dimensional induction machine model are calibrated according to evidence from measurements, by solving an inverse field problem. The set of parameters comprise material parameters as well as parameters that model three-dimensional effects. This allows a consideration of physical effects without explicit knowledge of its quantities. First, the accuracy of the approach is studied using an academic example in combination with synthetic data. Afterwards, it is successfully applied to a realistic induction machine model.

Index Terms—inverse problem, thermal model, induction machine, finite element method

I. INTRODUCTION

The trends for higher utilization and increased power density and therewith heat generation, sparks an interest in the accurate thermal simulation of electric machines [1], especially in highly constrained environments such as automotive applications. Numerous thin insulation layers of various materials [2],

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separating components on different electrical potentials, are prone to material failure due to overheating [3] or dielectric breakdown [4] which is a major cause of machine failure and limits the operating range. Their small thickness and their proximity to the windings, the main heat source, turn the particularly sensitive winding insulation into a component which necessitates thermal investigation [5]. Current thermal simulation is typically carried out with fast analytical circuit models based on lumped parameters or with the more complex and time-consuming numerical finite-element (FE) method [6]. The latter is more capable of handling complex geometries and boundary conditions (BCs) [7] and therefore more suited to analyse critical hot spots. Lumped parameter models are not capable of modelling localized effects, which are considered a critical aspect for diagnosing faults or failures. Considering three-dimensional (3D) models can be computationally demanding. Therefore, suitable approximations such as the reduction to two-dimensional (2D) models are employed [8]. In this work, the temperature distribution is simulated in a 2D FE model of a squirrel-cage induction machine. The 2D model is obtained by cutting the 3D model along its longitudinal cross-section. However, this simplified model omits relevant 3D effects, such as heat conduction in the axial direction and cooling by convection of the winding overhang [9]. A pragmatic approach for compensating these shortcomings is

manually fitting the 2D model parameters to measurement data [10]. The systematic introduction of data to enrich the model is a topic of ongoing research [11]. Here, poorly known parameters such as, e.g., homogenized electric or thermal conductivities, convection coefficients or some local thermal heat sources are calibrated on the basis of measurement results along an inverse problem solving step [12]. This approach is promising for a more precise identification of those poorly known parameters which are for example related to uncertainties in thermal material parameters, related to small and variable geometric details or related to variability in the heat sources.

This paper formulates an inverse thermal problem for model calibration. In the first step, the proposed approach is validated on academic models. For this, synthetic measurement data are used. Subsequently, the inverse problem is transferred to the machine model, first using synthetic- and then real measurement data obtained from a lab machine at TU Graz. Comparison against an independent measurement set validates the model calibration approach.

II. THERMAL MODEL

This section introduces the thermal problem, its discretization in FE analysis and homogenization as well as some remarks about 3D effects and homogenized model regions.

A. The static heat conduction equation

The static heat conduction equation describes the steady-state temperature distribution within the model domain Ω :

$$-\nabla \cdot (\lambda \nabla T) = g \quad \text{in } \Omega \subset \mathbb{R}^2, \quad (1a)$$

$$T = T_0 \quad \text{on } \Gamma_D, \quad (1b)$$

$$-\lambda \frac{\partial T}{\partial \mathbf{n}} = 0 \quad \text{on } \Gamma_N, \quad (1c)$$

$$\lambda \frac{\partial T}{\partial \mathbf{n}} + h(T - T_0) = 0 \quad \text{on } \Gamma_R. \quad (1d)$$

The solution, $T(\mathbf{x})$, is the temperature distribution, T_0 is the ambient temperature, $\lambda(\mathbf{x}, T)$ is the thermal conductivity, in general with spatial and temperature dependence, $h(\mathbf{x})$ is the heat-transfer coefficient and $g(\mathbf{x})$ is the internal heat source density term. The domain boundary is $\Gamma := \partial\Omega$ and is decomposed into Dirichlet Γ_D , Neumann Γ_N and Robin Γ_R boundary parts. The Dirichlet BC represents an isothermal coupling to the surrounding region. The Neumann BC describes perfect thermal isolation on the boundary surface. The Robin BC models thermal convection.

The field problem (1) may include a set of parameters θ that need calibration. Solving (1) to obtain a solution $T(\mathbf{x}; \theta)$ constitutes the forward problem and is denoted by $F(\theta)$. The forward problem is solved numerically by means of the FE method using the open-source solver `Pyrit` [13].

B. Finite-element discretization

The 2D cross-sectional model is meshed with triangles. The temperature

$$T \approx \sum_{j=1}^N u_j N_j \quad (2)$$

is discretized by first-order nodal shape functions $N_j(\mathbf{x})$ and degrees of freedom $u_j(\theta)$.

The Ritz-Galerkin [14] procedure leads to the system of equations

$$\mathbf{K}_\lambda \mathbf{u} = \mathbf{g}, \quad (3)$$

with the stiffness matrix and load vector

$$K_{\lambda,ij} = \int_{\Omega} \lambda \nabla N_j \cdot \nabla N_i \, d\Omega \quad (4a)$$

$$g_i = \int_{\Omega} g N_i \, d\Omega. \quad (4b)$$

C. 3D effects and homogenization

To reduce model complexity and computation times, a 2D model is simulated [15]. This model entails systematic errors that require consideration. Errors occur due to the neglect of 3D effects such as conduction in axial direction. Furthermore, thermal convection effects at the front and back machine side are not modelled [16]. Another approximation is introduced by inherently homogenizing small regions with similar thermal properties, which averages small-scale effects. [17]. These systematic errors are significantly reduced by introducing equivalent model parameters that are calibrated according to measurement data in the inverse problem.

III. INVERSE PROBLEM

The inverse problem reads

$$\begin{cases} \min_{\theta} J(\theta), \\ \text{such that (1) is fulfilled.} \end{cases} \quad (5)$$

Here,

$$J(\theta) = \sum_{i=1}^{N_{\text{samples}}} \sum_{j=1}^{N_{\text{sensors}}} (T_{\text{sim},i}(\mathbf{x}_j; \theta) - T_{\text{meas},i,j})^2 \quad (6)$$

is the cost function. $T_{\text{sim},i}(\mathbf{x}_j; \theta)$ are temperatures obtained by evaluating the solution of the parameterized forward problem $F(\theta)$ evaluated at the sensor position x_j and $T_{\text{meas},i,j}$ are the temperatures measured by the sensors. Index $j = 1, \dots, N_{\text{sensor}}$ and index $i = 1, \dots, N_{\text{samples}}$ count the sensors and the sampled operating points respectively. Therefore, (6) compares simulated and measured temperatures at discrete positions for various operating conditions.

Figure 1 outlines the iterative inverse parameter estimation process. The iteration is finished successfully after falling below a prescribed tolerance $\varepsilon = 10^{-6}$ or otherwise, is aborted if a maximum number of iterations is reached. The optimization method used in this study is the limited-memory-Broyden-Fletcher-Goldfarb-Shanno-bounded (L-BFGS-B) algorithm, as it is a standard optimizer available in scientific toolboxes. This algorithm is a bounded quasi-Newton algorithm using limited memory to approximate and update gradient information [18] to calculate the new set of parameters for each iteration (bottom left of Fig. 1) until the terminal criteria is met. Other optimization methods can be used as well.

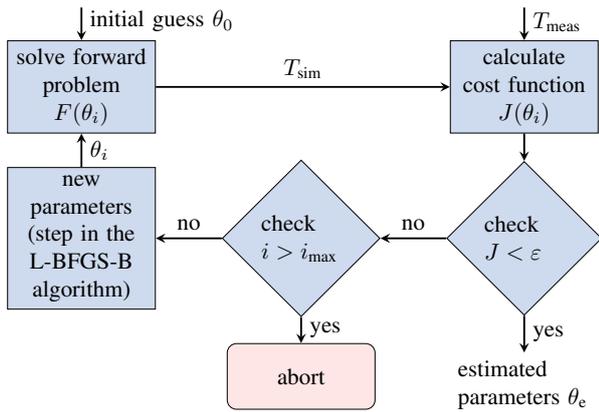


Fig. 1: Inverse parameter estimation process. Beginning with an initial guess θ_0 , the estimated parameters θ_e are obtained in an iterative process.

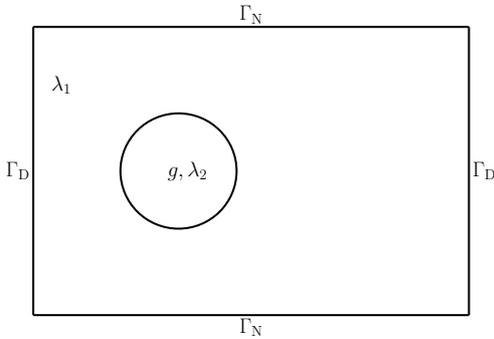


Fig. 2: Geometry of example 1. The model consists of a domain with thermal conductivity λ_1 and a domain with generated heat g and thermal conductivity λ_2 .

IV. ACADEMIC EXAMPLES

At first, the inverse parameter estimation is applied to two academic examples. The geometry of example 1 is depicted in Fig. 2. It consists of a rectangle with a combination of isothermal and adiabatic boundaries Γ_D and Γ_N . Additionally, a circular heat source is modelled. This example is used to investigate the solvability of the inverse problem and the accuracy of its solution. The geometry of example 2 is depicted in Fig. 3. In comparison to the first example, an additional domain surrounding the heat source and a number of discrete sensors are added. The additional domain models an insulation layer surrounding the heat source (i.e. winding). This example is used to investigate the sensitivity of the inverse problem with respect to the available temperature information.

For both examples, the thermal problem is solved with a parameter set $\theta = [\lambda_1, \lambda_2, \lambda_3]$. Synthetic measurement data are generated by solving the forward problem with the ground-truth values and afterwards perturbing the solution with noise. The noise is a standard distributed additive error with the standard deviation σ^2 . The noise is chosen to $\sigma^2 = 1$ K in

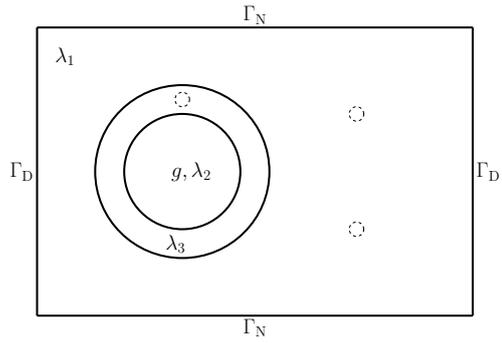


Fig. 3: Geometry of example 2 with discrete sensors, visualized by dotted circles. In addition to example 1, a cylindrical layer with thermal conductivity λ_3 is added.

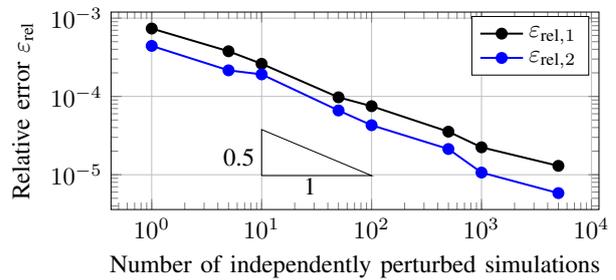


Fig. 4: Relative error of example 1 over number of independently perturbed simulation with multiple seeds.

accordance to the expected measurement noise. For example 1, these data are generated for each node of the FE mesh, whereas for example 2, only data for the nodes at the sensor positions are generated. To evaluate the accuracy of the solution of the inverse problem, the relative error is defined as

$$\varepsilon_{\text{rel},i} = \frac{|\lambda_{i,\text{estimated}} - \lambda_{i,\text{true}}|}{\lambda_{i,\text{true}}}. \quad (7)$$

A. Example 1: Rectangle with heat source

Figure 4 shows the relative errors $\varepsilon_{\text{rel},i}$ for an increasing number of independently perturbed simulations. The relative error of the estimation with 10 simulations ($N = 10$) is below 0.1% and decreases below 0.002% for $N = 10^3$. Therefore, the parameter estimation of example 1 is conducted successfully.

B. Example 2: Rectangle with insulated heat source

Example 1 proved the solvability of the inverse problem and accuracy of its solution, when measurement data are available on each node of the FE mesh. However, in reality the temperatures are only known at the locations of the sensors. Therefore, for example 2, the inverse problem is solved using only temperature information of the nodes associated with the discrete sensors. In this case, only 3 sensors are considered.

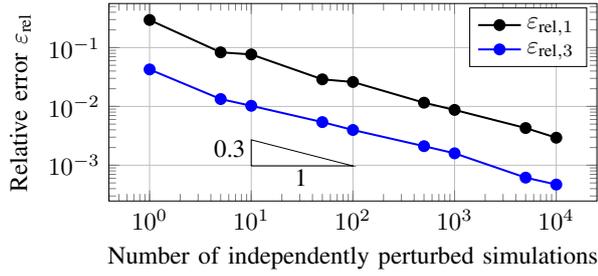


Fig. 5: Relative error of example 2 over number of independently perturbed simulations.

The relative error is 1% for 10 measurement samples and below 0.1% for $N > 10^3$ and converges as shown in Fig. 5.

Example 2 already reflects the nature of the upcoming technical example, i.e., the domains Ω_1 , Ω_2 and Ω_3 mimic the iron stator core, copper coil and epoxy resin insulation of the induction machine respectively. In the insulation layer, the sensitivity of the temperature to λ given as $\frac{dT}{d\lambda_i}(\mathbf{x})$ is maximal because $\lambda_3 \ll \lambda_1$ and $\lambda_3 \ll \lambda_2$. Therefore, a sensor is required in this region to evaluate this sensitivity and improve the parameter estimation of λ_3 . The resulting relative errors are small compared to the accuracy of the used sensors in the machine, the parameter estimation of the second model is also conducted successfully.

V. INDUCTION MACHINE

A. Machine data

Example 3 deals with a four-pole squirrel-cage induction machine [19] with a rated power of 3.7 kW at 1430 rpm. The 36 slots are filled with two-layer wire windings immersed in insulating epoxy resin. A sketch of the machine and its different components is given in Fig. 6. The temperature sensors are buried in the stator iron and the stator slots as shown in Fig. 7. For measurement purposes the machine is encased in a cooling jacket. The machine can be operated at various drive cycles [20] at a controllable temperature.

B. Measurement procedure

In the experimental setup, a constant electric power P is applied to the series connection of all windings of the machine at standstill. The resulting volumetric heat generated in the winding is

$$g = \frac{P}{V}, \quad (8)$$

where V is the volume of the copper winding. The heat source is generated due to Joule losses in the winding and depends on the winding resistance and current. During measurement, P or the cooling jacket temperature T_0 are increased in steps. A temperature sample is taken once a steady-state is observed from a limited change in temperature over time. A measurement run is shown in Fig. 8, where P is varied in

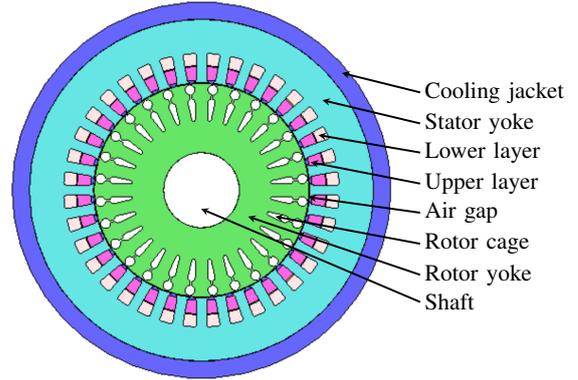


Fig. 6: Cross section of the induction machine. The stator and rotor yoke is laminated iron, the lower and upper layer combines the insulation system, which is epoxy resin and the windings, which are copper, the rotor cage is aluminium and the shaft is steel [10].

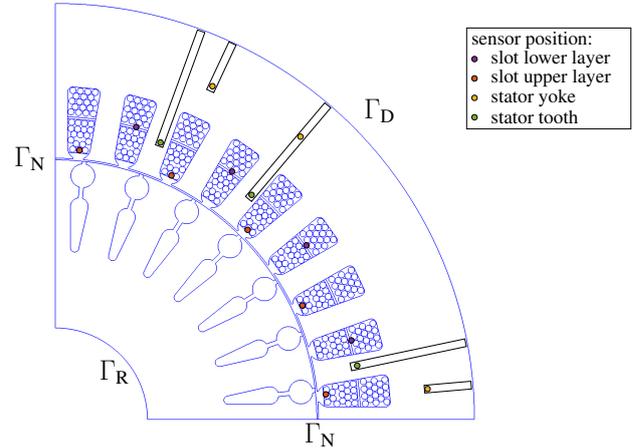


Fig. 7: Machine model including the position of the sensors, the applied boundary conditions and the resolved windings in the slot.

steps of 50 W, starting from 150 W, whereas T_0 is fixed to 25 °C. From this measurement run, six steady-state samples at four operation points are obtained. The dotted vertical lines indicate the points at which a steady-state is observed and a sample is taken. A sample contains measurement data from all sensors.

In total, measurement data are comprised of 20 samples at six different operating points. The operating points combine P between 150 W and 300 W and T_0 between 20 °C and 35 °C. Thanks to azimuthal symmetries, sensor data of the whole machine can be utilized, while only considering a quarter of the machine. Therefore, measurement data of 80 samples become available, which are split into two sets, 48 used for calibration and 32 used for independent validation. Either set contains a broad selection of operating points. The calibration

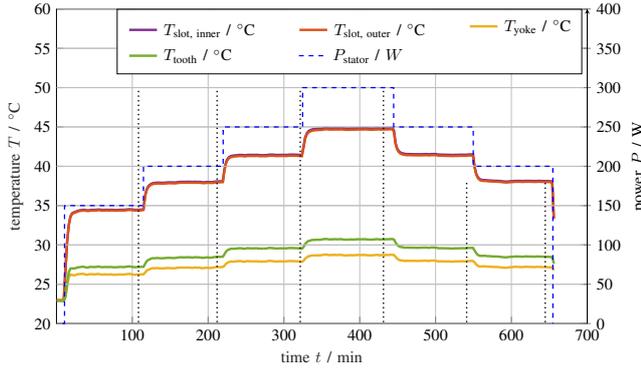


Fig. 8: Measured temperatures in the sensors (solid). The stator power is drawn in blue (dashed). The dotted vertical lines indicate six times, at which a steady-state is assumed and the samples are taken.

set is richer in order to achieve an accurate calibration.

C. Parameters selected for calibration

The parameters to be calibrated in the machine are $\theta = [\lambda_{\text{air}}, \lambda_{\text{insulation}}, \lambda_{\text{stator iron}}, \lambda_{\text{rotor iron}}]$. λ_{air} models conduction, convection as well as radiation effects in the air gap. $\lambda_{\text{insulation}}$ models the homogenized insulation consisting of several insulation layers such as winding enamels, layer separation and slot liners, as well as epoxy resin inclusions. Representing each layer individually raises the complexity of the model significantly due to different geometric scales by multiple orders of magnitude. A homogenization of these regions is expected to reduce the model complexity and speed up the computations, but the accuracy of the results needs to be verified. Exact material and homogenized properties are not known and are therefore estimated from experimental data. $\lambda_{\text{stator iron}}$ and $\lambda_{\text{rotor iron}}$ model thermal conduction through the stator and rotor lamination stacks.

In example 3, the results are calculated for a model that uses synthetic measurement data. Here, the quality of the estimation can be accurately assessed by comparing it to the available ground-truth values. In example 4, the actual measurement data are used on the model. The confidence in the estimation result can be trusted based on the previously learned estimation accuracy, but measurement and systematic uncertainties have to be kept in mind. The calibrated system is then compared to a set of validation measurements.

D. Example 3: Induction machine model with synthetic measurement data

The results for the inverse machine problem are shown in Tab. I for $N = 10$ and $N = 10^3$ samples of the synthetic measurements data. The principle of data generation is similar to the first examples, where the forward solution is calculated with ground-truth data. This solution is then perturbed N times independently. As in the example 1, the synthetic data for the whole machine are used for the inverse estimation. A good

estimation is achieved for all parameters except for $\lambda_{\text{rotor iron}}$. The measurements are insensitive to $\lambda_{\text{rotor iron}}$ because of the large thermal resistance of the air gap and because of the fact that no losses are introduced in the rotor. The conductivity is furthermore assumed to be equal to $\lambda_{\text{stator iron}}$ because the same material is used.

Next, similar to example 2, only synthetic data evaluated at the sensor positions are used. Here, a deterioration of the estimation of λ_{air} can be observed. A variation of its (already small) value can not be detected by the sensors.

TABLE I: Results of the inverse problem with synthetic measurements

Used Data	Parameter	Estimation results	
		$\varepsilon_{\text{rel}} (N = 10)$	$\varepsilon_{\text{rel}} (N = 1000)$
synthetic	λ_{air}	2 %	2 %
measurements	$\lambda_{\text{insulation}}$	0.5 %	0.25 %
	without sensors	$\lambda_{\text{stator iron}}$	0.07 %
	$\lambda_{\text{rotor iron}}$	50 %	50 %
synthetic	λ_{air}	100 %	10 %
measurements	$\lambda_{\text{insulation}}$	2 %	0.1 %
	with sensors	$\lambda_{\text{stator iron}}$	0.9 %
	$\lambda_{\text{rotor iron}}$	100 %	200 %

E. Example 4: Induction machine model with real measurement data

Table II lists the estimated parameters using all available calibration data ($N = 48$). According to the experience gained from example 3, $\lambda_{\text{rotor iron}}$ is no longer selected for calibration. Temperature gradients in the cooling jacket are observed as varying values for the nearby $\lambda_{\text{stator iron}}$ when simulating different sections of the machine. These differences are attributed to the good thermal conductivity of iron and the not perfectly symmetrical sensor positions. This influence is filtered out by averaging along the full circumference of the motor.

TABLE II: Results of the inverse problem using real measurement data.

Parameter	Estimation results	
	λ [$\frac{\text{W}}{\text{mK}}$]	for $N = 48$
λ_{air}	0.037	
$\lambda_{\text{insulation}}$	0.073	
$\lambda_{\text{stator iron}}$	30.70	

The error of the found solution is evaluated by comparing the results of a forward problem T_{cal} which uses the parameters of Table II with the validation set T_{val} . The relative error $\varepsilon_{\text{rel},T}$ is defined by

$$\varepsilon_{\text{rel},T,i} = \frac{|T_{\text{cal},i} - T_{\text{val},i}|}{T_{\text{val},i}}. \quad (9)$$

Averaging over all available validation data, relative errors of 0.64 % for the stator yoke, 0.34 % for the stator teeth and 0.77 % for the stator slots are obtained. Additionally, the relative error for all investigated operation points is below

the measurement accuracy of the used temperature sensors. Therefore, the calibration is considered successful and a good thermal model of the machine is acquired.

VI. CONCLUSION

2D FE models allow an efficient way to thermally characterize induction machines and predict hot spots. Neglecting 3D effects would lead to a poor accuracy. Here, 3D effects are inserted in the 2D model by calibrating a few of the model parameters according to available measurement data. The calibration is carried out by solving an inverse thermal field problem, enforcing the simulated temperature to be close to the temperatures measured by a set of sensors.

The paper successfully validates the approach for two academic examples and a third example being an induction machine model with synthetic measurement data. In a fourth example, it was shown that the method is capable of calibrating a 2D FE thermal induction machine model from measurement results, thereby including 3D effects.

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